Amendments to the Specification:

Please replace the paragraph [0035] of the published application with the following amended paragraph:

[0035] Fig. 1 shows diagrammatically a one-tube shock absorber 10 having an annular chamber 12 and a piston chamber 14 which chambers are separated by a piston 16. A freely floatable piston 18 separates the piston chamber 14 from a storage volume Vg which is filled for example with nitrog en ~~nitrogene~~ under a predetermined pressure. Suc h an arrangement is generally known. In Fig. 1 further a two-way-valve 20 ~~10~~ can be recognized having an integral valve spool or member 22. Two spaced annular grooves 24, 26 of the valve housing not shown are continuously connected with the annular chamber 12. Control edges 28, 30 of the valve member 22 co-act with the grooves 24, 26. In the constricted portion of the valve spool 20 a connection to the piston chamber 14 is established. A control piston 32 - which is a differential piston — is connected to valve spool 22. Its larger effective surface 34 is subject to the pressure of the piston chamber 14. A smaller effective surface 36 is subject to the pressure of the annular chamber 12. A spring 38 acts on control piston 32, and a spring 40 acts on the right end of the valve spool. The springs 38, 40 are designed such that the valve spool is in the shown neutral position upon static balance at the shock absorber 10.

Please replace the paragraph [0059] of the published application with the following amended paragraph:

[0059] The function of the invention is to be illustrated along fig. 10 for a two-tube shock absorber. The outer tube is not shown, rather substituted by storage 56 which is shown in fig. 3. The storage is connected to piston chamber 14 through check valve RS_B. The piston chamber is also connected to annular chamber 12 through a check valve RS_K. A first piston valve VK1 and a second piston valve VK2 are parallel connected. The function of the piston valves VK1 and VK2 correspond to that described in connection with figs. 5 and 6. The differential pressure between annular chamber 12 and piston chamber 14 acts on the piston arrangement of piston

valve VK2. This represents the dampening force. The piston arrangement of the piston valve VK1 represents the spring force. This has been described above. The bottom valve V_B in fig. 10 resembles valve arrangement of fig. 4 with respect to its structure. The right effective surface is subject to the compensation pressure P_{aB} while the right effective surface of the control piston arrangement of piston valve VK1 is subject to compensation pressure P_{ak}. The left effective surface of piston valves VK1 and VK2 is in communication with the piston chamber 14 through orifices R_{hDI} and R_{hD2}. The compensation or balancing pressure P_{aB} is connected through orifice R_{hDB}.